

AMENDMENTS TO THE SPECIFICATION

Please amend the paragraph beginning on Page 2, Line 4 and ending on Page 2, Line 29 as indicated below. Note that this paragraph corresponds to paragraph [0004] from U.S. Patent Application Pub. No. 2006/0103074.

This object is achieved by means of a secondary sealing element consisting of a base body of a synthetic material and comprising a base portion and a seal portion. The base and seal portions each ~~includes~~ include a through bore for the passage of a component, said through bores are axially adjacent and coaxially aligned with each other. An annular disc element is accommodated in said base portion and includes a through bore coaxially aligned with the through bores in the base and seal portions. The annular disc element is made of a material which differs from that of the base body. According to the invention, the through bore of the annular disc element has a radial dimension which, in the unloaded state, is greater than that of the through bore in the seal portion and smaller than that in said base portion of the base body, and in that the material of the annular disc element comprises a carbon material. At low pressures of the medium, there is practically no load on the annular disc element and the seal portion of the base body consisting of the flexible synthetic material performs the sealing function practically alone. The sealing forces arising thereby can easily be optimised in ~~regarding~~ regard to adequate movement of the component concerned (seal ring). By contrast, at higher pressures of the medium, the sealing function is shifted more and more onto the ring seal element which is now pressed into sealing engagement with the surface requiring sealing. A gap between the components requiring sealing is thereby closed at the same time. Because of the good tribological properties of the carbon material, the movement of the component concerned (seal ring) is maintained to an adequate extent despite the firm engaging relationship between the ring seal element and the surface. Under high pressure conditions, the load on the seal portion of the base body may be relieved entirely. In accordance with a further development of the invention, the secondary sealing element can be provided in a recess in an end face of the base

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portion and project axially beyond the face. The secondary sealing element thereby simultaneously produces a radial and an axial sealing effect.

Please amend the paragraph beginning on Page 5, Line 24 and ending on Page 6, Line 26 as indicated below. Note that this paragraph corresponds to paragraph [0017] from U.S. Patent Application Pub. No. 2006/0103074.

Another feature of the ring seal element 25 is that it contains a through bore 30 through which the sleeve 18 can be inserted with suitable play in the unloaded state of the ring seal element 25. The through bore 30 is coaxial with the through bore 17 of the base portion 13 and has a radial dimension d which is smaller than the radial dimension D_1 of the base portion 13 in the unloaded state of the secondary sealing element 10. A further feature of the ring seal element 25 is that the radial dimension d is larger than that D_2 of the through bore 31 of the seal portion 14 in the unloaded state. As can also be taken from FIGS. 1 and 2, the pressure of a medium requiring sealing, wherein this is preferably, but not exclusively, a gas, is effective both radially and axially on the ring seal element 25 in that the medium can penetrate into the gap (shown enlarged to an exaggerated extent in the drawing) between the outer periphery of the ring seal element 25 and the inner periphery of the recess 11 and also between the seal ring 3 and the end face 26. As a consequence thereof, the ring seal element 25 experiences a radial upsetting which becomes greater as the pressure of the medium increases. Thus, as the pressure of the medium increases, the ring seal element 25 is pressed into ever firmer sealing engagement with the neighbouring surface of the sleeve 18 in order to seal the radial ~~gap~~ gaps between the seal ring 3 and the sleeve 18. The seal portion 14 is thereby practically freed of pressure. By contrast, for smaller pressures of the medium, the sealing is effected primarily by the seal portion 14 due to the engagement between the radially inner sealing surface 23 thereof and the surface of the sleeve 18 since here, the contact pressure on the ring seal element 25 is not sufficient to obtain an adequate sealing effect due to the above-mentioned radial dimensions of the through bores 30, 31. The radial bias force with which the sealing surface 23 is pressed against the surface of the sleeve 18 can be set in such a way that the axial movement of the thrust ring is not substantially impaired, i.e. defined radial sealing forces are present which do not cause an unwanted "hang-up effect" at low pressures of the medium. This freedom of movement is also ensured at high

pressures of the medium because of the good tribological properties of the carbon material from which the ring seal element 25 is formed. An effect can likewise be had on the freedom of movement of the ring seal element 25 by suitably dimensioning the axial dimensions thereof. At the same time, the ring seal element 25 prevents the danger of a flow or extrusion of the material of the base body 5 into the ~~gap~~-sgaps under high pressure conditions.